

## Chapter 4

**4-1** Let  $x_1 = x$ ,  $x_2 = y$ ,  $u_1 = u$ , and  $u_2 = v$  in Eq. (4-17). Then

$$\text{Equation, } x \text{ - direction} \quad \frac{\partial P}{\partial x} = \frac{\partial^2 u}{\partial x^2}$$

$$\text{Equation, } y \text{ - direction} \quad \frac{\partial P}{\partial y} = 0$$

Thus,  $P = P(x)$  and  $\frac{\partial P}{\partial x} = \frac{dP}{dx}$ . The equation becomes

$$\frac{dP}{dx} = \mu \frac{d^2 u}{dy^2}$$

**4-2** Consider the continuity equation (4-5) in cylindrical coordinates with  $\nabla \cdot \mathbf{V}$  defined in Appendix D,

$$\nabla \cdot \mathbf{V} = \frac{1}{r} \frac{\partial}{\partial r} (r V_r) + \frac{1}{r} \frac{\partial V_\phi}{\partial \phi} + \frac{\partial V_x}{\partial x} = 0$$

where  $V_r = v_r$ ;  $V_\phi = v_\phi$ ; and  $V_x = u$ . Let  $r = R + y$  and  $x = R\phi$ , where  $R$  is the radius of the inner surface. Note that  $\partial/\partial x = 0$  (no axial flow). Thus

$$R \frac{\partial v_r}{\partial y} + y \frac{\partial v_r}{\partial y} + v_r + R \frac{\partial v_\phi}{\partial x} = 0$$

For  $v_r$  small and  $y \ll R$ , the middle two terms in the equation are negligible compared to the first and last terms. Thus the equation becomes identical to (4-7) in Cartesian coordinates. (Actually these are curvilinear coordinates with  $y \ll R$ ). Using similar arguments, the applicable momentum equation is (4-11), with the convective acceleration terms neglected. The applicable energy equation is (4-38).

**4-3** The appropriate set of coordinates is the fixed, non-rotating cylindrical system with  $r$ , the radial direction;  $\phi$ , the circumferential direction; and  $x$ , the axial direction above the disk. In the governing equations, all three velocity components will appear, but derivatives with respect to  $\phi$  will be zero due to rotational symmetry.

The flow over a rotating disk is boundary layer in character, but the complete Navier-Stokes equations can be solved in exact form (see Schlichting, ref. 2, page 93). The applicable energy equation is (4-35) with the conduction gradient in the  $r$ -direction neglected. (See Schlichting, page 296, for references to heat transfer solutions.)

**4-4** The applicable energy equation is (4-31). Assume no mass diffusion. Use the definition of enthalpy,  $i = e + P/\rho$ , and let  $\rho$  be constant, yielding

$$\rho \frac{De}{Dt} - \nabla \cdot k \nabla T = S$$

For no fluid motion, the substantial derivative reduces to

$$\rho \frac{\partial e}{\partial t} - \nabla \cdot k \nabla T = S$$

This is the classic heat conduction equation for a solid where the thermal equation of state is  $de = cdT$ . For steady conduction and constant properties, the Poisson form of the conduction equation is obtained, and when  $S$  is equal to zero, the Laplace equation is obtained.

$$\nabla^2 T = \frac{S}{k}$$

**4-5** Eq. (4-1) can be considered a continuity equation for the  $j$  - component of a mixture, but the creation term must be added, resulting in

$$\frac{\partial G_{tot,j,x}}{\partial x} + \frac{\partial G_{tot,j,y}}{\partial y} = \dot{m}_j''$$

Now substitute  $G_{tot,j} = G_{diff,j} + G_{conv,j}$ , where  $G_{conv,j} = m_j G$  and  $G_{diff,j} = -\gamma_j \nabla m_j$ . (Recall that  $G$  is the total mass flux vector.) Ignore  $G_{diff,j,x}$  as a boundary layer approximation and substitute into the continuity equation.

$$m_j \left[ \frac{\partial G_x}{\partial x} + \frac{\partial G_y}{\partial y} \right] + G_x \frac{\partial m_j}{\partial x} + G_y \frac{\partial m_j}{\partial y} - \frac{\partial}{\partial y} \left( \gamma_j \frac{\partial m_j}{\partial y} \right) = \dot{m}_j''$$

**4-6** For conservation of mass, the terms for mass flow rates (inflow on the radial face and the axial face) are

$$\begin{aligned}\dot{m}_r &= G_r A_r & G_r &= \rho v_r; & A_r &= r d\phi dx \\ \dot{m}_x &= G_x A_x & G_x &= \rho u; & A_x &= r d\phi dr\end{aligned}$$

Application of the procedure leading up to equation (4-2), assuming steady flow, yields

$$\frac{\partial(u\rho)}{\partial x} + \frac{1}{r} \frac{\partial}{\partial r}(r\rho v_r) = 0$$

Note the appearance of  $r$  comes from dividing through the equation by the differential volume, ( $r d\phi dx$ ). For constant properties, Eq. (4-9) is found.

$$\frac{\partial u}{\partial x} + \frac{1}{r} \frac{\partial}{\partial r}(r v_r) = 0$$

For  $x$ -momentum, the  $x$ -momentum flow rates, inflow on the radial face and the axial face, and the  $x$ -forces are

$$\begin{aligned}\dot{M}_r &= u \dot{m}_r = u(G_r A_r) \\ \dot{M}_x &= u \dot{m}_x = u(G_x A_x)\end{aligned}$$

and

$$F_x = -\sigma_x A_x - \tau_{rx} A_r \quad \sigma_x \approx -P \quad \tau_{rx} = \mu \frac{\partial u}{\partial y}$$

Application of the procedure leading up to Eqs. (4-10) and (4-11), assuming steady flow and constant properties yields

$$\rho u \frac{\partial u}{\partial x} + \rho v_r \frac{\partial u}{\partial r} = -\frac{dP}{dx} + \mu \frac{1}{r} \frac{\partial}{\partial r} \left( r \frac{\partial u}{\partial r} \right)$$

For energy, the energy flow rates, inflow on the radial face and on the axial face, and the corresponding heat and work rates are

$$\begin{aligned}\dot{E}_r &= \left( i + \frac{1}{2} u^2 \right) \dot{m}_r = \left( i + \frac{1}{2} u^2 \right) G_r A_r \\ \dot{E}_x &= \left( i + \frac{1}{2} u^2 \right) \dot{m}_x = \left( i + \frac{1}{2} u^2 \right) G_x A_x\end{aligned}$$

and

$$\dot{q}_r = -k A_r \frac{\partial T}{\partial r} \quad \dot{q}_x = -k A_x \frac{\partial T}{\partial x} \quad \dot{W}_{shear,r} = u \tau_{rx} A_r$$

Application of the procedure leading up to Eqs. (4-27) and (4-28), assuming steady flow, no mass diffusion, and constant properties yields

$$\rho u \frac{\partial i}{\partial x} + \rho v_r \frac{\partial i}{\partial r} - \frac{1}{r} \frac{\partial}{\partial r} \left( kr \frac{\partial T}{\partial r} \right) - \mu \left( \frac{\partial u}{\partial r} \right)^2 - u \frac{dP}{dx} = 0$$

and for ideal gases and incompressible liquids, using the approximations similar to the development leading to Eq. (4-38), but retaining the pressure gradient term the enthalpy equation reduces to

$$u \frac{\partial T}{\partial x} + v_r \frac{\partial T}{\partial y} - \alpha \frac{1}{r} \left[ \frac{\partial}{\partial r} \left( r \frac{\partial T}{\partial r} \right) \right] - \frac{\text{Pr}}{c} \left( \frac{\partial u}{\partial r} \right)^2 - \frac{1}{\rho c} \frac{dP}{dx} = 0$$

**4-7** These steps are basically the redevelopment of the formulation leading up to Eq. (4-39).